

**UNITED STATES PATENT APPLICATION
FOR GRANT OF LETTERS PATENT**

**Jack Simpson
Inventor**

**RESILIENT SCRAP STRIPPER
FOR A CORRUGATED BOARD
ROTARY CUTTING DIE**

85E040" 59545060

RHODES, COATS & BENNETT, L.L.P.
909 Glenwood Avenue
P.O. Box 5
Raleigh, NC 27602
(919) 832-3946

SECRET

Rotary cutting dies of the type described above are often employed to produce slots or various shaped openings in the blank sheet of corrugated board material that is being processed. As such, provisions for removing or stripping the severed scrap material from certain cutting blades and the processed blank must be

provided. Otherwise, if not actively removed from the vicinity of the cutting die, the scrap material tends to collect around the cutting blades and, render the rotary cutting die inoperable.

Equally important, with regard to the stripping or ejection of scrap material, is the direction in which the successfully stripped or ejected scrap exits the cutting die apparatus. As the usable product of the cutting process is typically expelled directly outward from the nip of the rotary cutting die apparatus, it is desirable, in order to obtain complete separation of scrap and corrugated board product, that the scrap be ejected from the rotating cylinders at a significantly different trajectory than the corrugated board product.

In the past, resilient rubber strips or pads made of closed cell, high density foam or gum rubber have been placed adjacent the cutting blade so as to forcibly eject the corrugated board scrap material. However, previous resilient strippers have suffered from a number of shortcomings, particularly with regard to directional control of the ejected scrap material. First, it should be noted that scrap strippers of the prior art do not typically extend substantially past adjacently disposed scrap cutting blades even when in a non-compressed posture. Thus, while these scrap strippers may possess enough resiliency and strength to strip cut scrap pieces from the adjacent blades, they do not have the ability to significantly play a major role in controlling the direction and flight of scrap pieces exiting the nip between the rotating cutting die and the anvil. Thus, one typically finds cut

scrap flying outwardly and sometimes upwardly out of the nip. The net result is that the scrap becomes airborne and intermingled with the exiting corrugated board product and ultimately becomes packaged with the supposedly clean product. Obviously, scrap intermixed with the final corrugated product is most undesirable, especially in certain industries and certain cases such as with pizza containers, for example.

Therefore, it has been found that conventional scrap strippers are incapable of extending or achieving a height that enables them to exercise control over the flight or trajectory of the exiting scrap pieces. Again, this is because in typical applications, the relatively hard rubber stripper pads when expanded (non-compressed) just barely extends in height past the adjacent scrap blade or blades and, as such, they cannot, in reality, significantly affect the flight path of the scrap pieces exiting the nip.

Therefore, there remains a need for a practical, reliable, and cost effective resilient scrap stripping member for use with corrugated board rotary cutting dies which efficiently separates severed scrap material from an associated blank of corrugated board material and which furthermore provides control of the ejected or stripped scrap trajectory as it exits the rotary cutting die apparatus.

SUMMARY OF THE INVENTION

09054565-040398

The present invention recognizes that in order for a resilient foam or rubber scrap stripper to provide effective directional control of the ejected scrap material, they must act so as to hold the cut scrap against the associated anvil at least momentarily as the cut scrap pieces exit the nip between the rotary cutting die board and the anvil. By doing so, the scrap will be ejected by the rotating anvil in a direction that is significantly different from the direction of the ejected product. That is, by holding the severed scrap material against the downwardly rotating anvil, the scrap will tend to be ejected from the die assembly at a trajectory that is angled significantly lower than the exit trajectory of the processed corrugated board product. The ability to consistently and reliably ensure that the cut scrap material will exit the die assembly with a significantly lower trajectory than the product material, results in an improved, efficient and cost-effective die cutting operation.

To achieve this, the present invention provides a resilient rubber type scrap stripper that includes a base that is secured to the die board and an angled flexible finger integral with the base and extending outwardly therefrom. The finger, in a non-compressed position, extends at an angle outwardly past the terminal edge of any adjacently disposed scrap blades. During the die cutting operation, the scrap stripper is disposed between the rotary cutting die and the corrugated board

product being passed between the anvil and the rotary cutting die. As the scrap stripper enters the nip between the die board and the anvil, the entire stripper, including the base and the angled finger, is compressed such that adjacently disposed scrap blades cut selected pieces of scrap from the corrugated board blank passing through the nip. As the scrap stripper exits the nip, it finds itself still disposed between a cut scrap piece and the cutting die. However, as the scrap stripper exits the nip, the same will expand and as it expands, it will strip the underlying piece of cut scrap from the adjacent blade or blades. In addition, the angled finger that forms a part of the scrap stripper will tend to extend and in this process, the flexible angled finger will engage and push the cut scrap piece against the underlying and rotating anvil. The ability of the angled finger to extend substantially beyond the height of the scrap blade or blades, enables the flexible finger to hold the cut scrap piece against the anvil on the downstream side of the nip. This, it follows, causes the cut scrap to be separated from the corrugated board product and to be directed generally downwardly alongside the downstream side of the anvil.

It is therefore an object of the present invention to provide a rotary cutting die that will effectively separate cut scrap from corrugated board product.

Still a further object of the present invention is to provide a scrap stripper for a rotary cutting die that is extendable substantially past the height of an adjacent scrap cutting blade for holding exiting scrap pieces against a cooperating

anvil in order that the anvil may effectively control the flight or trajectory of the scrap piece exiting the nip between the die board and the anvil.

Other objects and advantages of the present invention will become apparent and obvious from a study of the following description and the accompanying drawings which are merely illustrative of such invention.

09054565.040393

BRIEF DESCRIPTION OF THE DRAWINGS

Figure 1 is a perspective view of a corrugated board rotary die cutting apparatus which incorporates resilient scrap strippers of the type contemplated by the present invention.

Figure 2 is a perspective view of the resilient scrap stripper of the present invention.

Figure 3A is a side elevational view of the resilient scrap stripper of the present invention.

Figure 3B is a front elevational view of the resilient scrap stripper of the present invention.

Figure 3C is a rear elevational view of the resilient scrap stripper of the present invention.

Figure 4A is a partial sectional view of a corrugated board rotary die cutting apparatus incorporating the resilient scrap strippers of the present invention and which illustrates the relative positioning and orientation of the scrap strippers and incoming blank of corrugated board material prior to cutting.

Figure 4B is a partial sectional view similar to Figure 4A but which illustrates a leading scrap stripper entering the nip between the rotary cutting die and the anvil.

Figure 4C is another sequence view illustrating a pair of scrap strippers being compressed between the die board and a cut piece of scrap as the scrap strippers move through the nip.

Figure 4D is another sequence view of the rotary die cutting apparatus incorporating the resilient scrap strippers of the present invention, which illustrates the recoiling action of a trailing scrap stripper as it holds the severed scrap against a downwardly rotating anvil cylinder.

Figure 4E is another sequence view of the rotary die cutting apparatus incorporating the resilient scrap strippers of the present invention, which illustrates the further recoiling action of the trailing stripper as it continues to hold the severed scrap against a downwardly rotating anvil cylinder following clearance of the lead scrap stripper.

Figure 4F is a final sequence view showing the trailing scrap stripper engaging a trailing portion of the severed scrap piece and continuing to hold the trailing portion of the scrap against the anvil.

Figure 5 is a fragmentary schematic view of a rotary cutting die showing an alternative embodiment for the scrap stripper of the present invention.

DETAILED DESCRIPTION OF THE INVENTION

With further reference to the drawings, Figure 1 illustrates a rotary die cutting apparatus, indicated generally by the numeral 40, for cutting corrugated board CB. The rotary die cutting apparatus 40 basically comprises a pair of

rotatably mounted, cooperating cylinders or drums. More particularly, the assembly includes a cutting cylinder 50 and an anvil cylinder 60. Cutting cylinder 50 is at least partially surrounded or sheathed with a generally cylindrical die board or base 52. Secured around the anvil 60 in conventional fashion is a layer of neoprene rubber 60a, or other suitable material, against which the die board 52 cuts.

Typically, the cutting cylinder 50 and anvil cylinder 60 are disposed closely adjacent each other so as to define a nip or nip area 64 between the cylinder and anvil. In a conventional corrugated board die cutting operation, the cutting cylinder 50 and anvil cylinder 60 are driven at essentially the same speed and sheets of corrugated board CB are fed through the nip 64. As the corrugated board CB is fed through the nip, the ~~cutting die~~ *cylindrical die board* 52 cuts through the corrugated board and against the outer cylindrical sheet of neoprene rubber 60a secured to the anvil cylinder 60. Thus in conventional fashion, the sheets of corrugated board are trimmed, scored, slitted, etc. so as to produce a sheet or blank of corrugated finished product along with cut scrap.

In order to produce the corrugated board product, the rotary cutting die ^{*board*} 52 is typically provided with a series of knives or blades and scoring rules that trim, cut and score selective areas of the corrugated board fed into and through the nip 64.

Note in Figure 1, for example, that the cylindrical die board 52 does include various ~~cutting~~ ^{*board*} blades and/or scoring rules. In addition, the cutting die 52 includes

various trim and scrap strippers and could include product ejectors as well (It should be pointed out that the die board 52 in Figure 1 does not show product ejectors but that such are typically incorporated into die board designs). The trim and scrap strippers typically function to strip cut trim or scrap from adjacently disposed cutting blades. The product ejectors act to eject the final corrugated product from certain blades that extend from the cutting die 52.

The present invention deals with scrap strippers. More particularly, the die board 52 is provided with a series or array of scrap strippers, some of which are aligned along opposed outer sides of the die board 52 while a group of nine such scrap strippers are disposed within the confines of a rectangular blade network that is effective to cut a rectangular slot or opening from a sheet of incoming corrugated board. Figures 2 and 3a-3c show in detail the design of the scrap strippers secured to the cutting die 52. In these drawings, the scrap stripper is indicated generally by the numeral 10.

Viewing the scrap stripper 10 in more detail, it is seen that the same assumes a generally V-shape and includes a base 12 and a flexible angled finger 22. Base 12 includes a pair of vertical sides 14, a horizontal bottom or mounting surface 16, a generally horizontal inner surface 18, and an angled rear edge surface 20. The flexible finger 22 includes a pair of vertical sides ²⁴~~20~~, an angled contact surface 26, an angled inner surface 28 and an angled rear edge surface 30. The base 12 and flexible finger 22 are joined at an angle so as to form a generally V-

shaped structure. In fact, in the embodiment illustrated herein, the finger 22 and base 12 merge and together they form a leading or forward edge 32. A generally wedge-shaped space or relief area is formed between the base 12 and the angled flexible finger 22.

As seen in Figure 3A, finger 22 and base 12 form an angle that is referred to as angle A. Angle A in one preferred embodiment is approximately 30-75 degrees. But it should be appreciated that angle A could vary depending on the application and structural characteristics of the material from which the stripper 10 is constructed.

It should be appreciated at this point that, although the stripper 10 is described herein in terms of a discrete base segment 12 and a discrete finger segment 22, in practice the stripper 10 would typically be fabricated utilizing a one-piece molding or machining type process, such that these segments are continuous. Furthermore, in the embodiment contemplated herein, the stripper would typically be fabricated of a 25 - 60 durometer closed cell rubber polymer, such as neoprene, although other materials exhibiting satisfactory elastic or resilient properties could be employed. Preferably, it is believed that a durometer of approximately 40 would impart to the stripper sufficient hardness and flexibility to perform its intended functions.

With particular reference to the sequence of drawings illustrated in Figures 4A-4F, a pair of scrap strippers 10 are shown therein and the sequence of views

illustrates the movement of the strippers along with a sheet of corrugated board CB through the nip 64 defined between the cutting cylinder 50 and the anvil 60. To illustrate the operation of the scrap strippers 10, only two are shown in Figures 4A-4F. Along with the pair of strippers 10, there is shown a pair of scrap cutting blades 54 and 56. Cutting blades 54 and 56 function to cut a piece of scrap S from the corrugated board CB being fed into the nip 64. It will be appreciated by those skilled in the art, that the scrap strippers 10 can be employed individually on the die board 52 or can be employed in groups such as with the group of nine strippers shown about the left-hand side of the die board 52 in Figure 1. Further, the scrap cutting blades could surround the one or more scrap strippers 10 to effectively cut a hole, notch or even a trim piece from the corrugated board CB passing through the nip 64. Again, for purposes of illustration, in the sequenced views of Figures 4A-4F, only two scrap cutting blades are shown, the leading scrap blade 54 and the trailing blade 56.

Now, viewing Figure 4A particularly, it is seen that the corrugated board CB has been fed into the nip 64 and the two scrap strippers 10 are being turned clockwise and are about to approach the nip 64. In Figure 4A, neither blade 54 or 56 has engaged the corrugated board CB but the finger portion of the lead scrap stripper 10 has engaged the corrugated board CB and has started to flex inwardly towards the base of the stripper. In the case of the trailing scrap stripper 10, the

finger appears to be fully extended but has just reached the point where the outer terminal end of the finger has engaged the corrugated board CB.

In Figure 4B, the lead scrap blade 54 has engaged and cut through the corrugated board CB. The lead scrap stripper 10 has been further compressed between the die board 52 and the corrugated board CB to such an extent that the flexible finger portion lies adjacent the base. In addition, with respect to the trailing scrap stripper 10, the finger portion has fully engaged the corrugated board CB and has been flexed back and inwardly to an intermediate compressed or retracted position. Trailing scrap blade 56 remains spaced from the corrugated board CB.

Turning to Figure 4C, the lead scrap blade 54 has completed its cutting operation and has disengaged itself from the corrugated board CB. The finger of the lead stripper 10 has started to exert a downward or outer force on the underlying scrap piece S and, as viewed in Figure 4C, has already acted (alone or in concert with other strippers) to strip the scrap piece S from the lead scrap blade 54. The trailing scrap stripper 10 is disposed squarely within the nip 64 defined between the cutting cylinder 50 and the anvil 60. It is substantially compressed between the die board 52 and the scrap piece S. At the same time, the trailing scrap blade 56 has rotated sufficiently clockwise to engage the corrugated board CB and to cut through the same so as to separate the rear portion of the scrap piece S from the trailing portion of the corrugated board CB.

005455-040398

In Figure 4D, it is seen that the cutting cylinder 50 and anvil 60 have rotated clockwise to a point where both blades 54 and 56 are separated from the corrugated board CB and there is produced a cut scrap piece S.) Also at this point, the leading scrap stripper 10 has assumed its normal erect or extended position and is completely separated from the corrugated board CB and the scrap piece S. However, the trailing scrap stripper 10 remains engaged with the scrap piece S and acts to hold the scrap piece S against the rubber layer 60a formed about the outer portion of the anvil 60. In particular, the finger portion of the trailing stripper 10 has moved and flexed outwardly from the position shown in Figure 4C, and while the finger portion of the trailing stripper has not reached a fully extended position, it nevertheless lies flush against a trailing or rear portion of the scrap piece S) and holds the scrap piece S) against the rotating anvil 60. Thus it is appreciated that as viewed in Figure 4D that the cooperative actions of the scrap strippers at this point in the cycle have already started to separate the scrap piece S from the corrugated board product that is being directed generally horizontally and outwardly from the nip 64 defined between the cutting cylinder 50 and the anvil 60.

Turning to the next sequence drawing, Figure 4E shows that the cutting cylinder 50 and the anvil 60 have both advanced from the position shown in Figure 4D. However, the flexible finger of the trailing scrap stripper 10 still remains in contact with the trailing portion of the scrap piece S.)

Finally, Figure 4F shows the trailing scrap stripper 10 being disposed in a fully erect or extended position but with the terminal end of the flexible finger still being engaged with the trailing portion of the cut scrap piece S. It is thusly seen from Figure 4F, that the trailing scrap stripper 10 acts to continuously engage the cut scrap piece S and to hold the scrap piece against the anvil 60 for a significant period, even after the trailing stripper 10 has exited the nip 64. This results in the scrap piece S being held against the anvil 60 for a certain period of time after the scrap piece S has exited the nip 64. Consequently, by holding the scrap piece S against the anvil 62, the cut scrap pieces are directed generally downwardly adjacent the downstream side of the anvil and in the process, the scrap pieces are separated from the corrugated board product which typically is ejected along a generally horizontal path from the nip 64.

From the foregoing specification and discussion, it is appreciated that the scrap stripper 10 performs two basic functions. First, the stripper 10 acts to strip cut scrap pieces S from one or more adjacently disposed knife or blade segments. Secondly, because of the configuration of the stripper 10 and its ability to extend substantially past the height of any adjacent blades, the stripper through the flexible finger 22, acts to engage the cut pieces of scrap S and to hold the scrap pieces against the anvil 60 as the scrap pieces move out of the nip. Due to the extension of the finger 22, the scrap pieces S are held against the anvil for a substantial period after the scrap pieces S have been cut.

By holding the scrap pieces S against the anvil, it follows that the scrap pieces acquire a significant downward velocity component that directs the scrap downwardly adjacent the downstream side of the anvil. Thus, the strippers 10 generally assure that the scrap pieces are not directed horizontally out of the nip which, in such case, the scrap would become intermingled and mixed with the finished corrugated board product produced by the rotary cutting die assembly 40.

Further, the design of the stripper 10 allows it to perform both of the above functions effectively and efficiently. Because of the flexible nature of the finger 22, it can be pressed or pushed backwards, relative to the direction of travel of the die board 52, against the base 12 of the stripper and both the finger and base can be compressed simultaneously, resulting in the storing of energy in the compressed stripper. Once released, both the finger 22 and base 12 can expand or recoil with sufficient energy and force to push the cut scrap pieces S from any adjacent blades. Thereafter, the finger 22 can continue to extend and maintain contact with the underlying scrap piece S and, in the process, hold the scrap piece S against the anvil 60 such that the anvil will act to separate the scrap from the corrugated board product.

Turning to Figure 5, there is shown therein an alternative design for the scrap stripper. This alternative design comprises a resilient scrap stripper which is indicated generally by the numeral 78. Scrap stripper 78 is, like the scrap stripper discussed above, constructed of a compressible resilient material. Basically scrap

stripper 78 comprises a base 80 and an outer flexible portion that is indicated generally by the numeral 90. In the embodiment of Figure 5, the outer flexible portion 90 includes a pair of legs 82 and 84 that extend outwardly from the base and meet to form an apex 86. Defined within the confines of the base 80 and the legs 82 and 84 is a relief area 88.

Scrap stripper 78 (Figure 5), like the scrap stripper 10 shown in Figure 2, is designed to be mounted to the board or base 52 that forms a part of the rotary cutting die. In particular, as illustrated in Figure 5, base 80 is designed to be secured by glue or other suitable means to the board or base 52. In addition, the scrap stripper 78 is adapted to be situated adjacent or between a series of knives 54 and 56. It is important to appreciate that the scrap stripper 78 functions substantially the same as the scrap stripper 10 discussed above and shown in Figures 1 through 4F. In particular, the outer flexible portion 90 of the stripper is designed such that it can be compressed between the rotary cutting die and anvil as it passes through the nip defined between the cutting die and the anvil. In particular, as viewed in Figure 5, the forward leading leg 82 would tend to be compressed towards the die board 52 in a direction generally opposite to the direction of rotation of the cutting die and the anvil. Effectively, the relief area 88 defined within the scrap stripper 78 permits the outer portion 90 to be compressed to where the entire scrap stripper 78 would lie in a compressed state within the nip between the cutting die and the anvil. As the scrap stripper 78 exits the nip, the

leading leg 82, along with the trailing leg 84, would tend to expand and project outwardly from the base and in that process would engage a cut piece of scrap and push the same against the anvil in the same manner discussed above with respect to the scrap stripper 10 shown in Figure 2.

Also, the efficiency of the scrap strippers disclosed herein may be enhanced by selectively weighting certain portions of the strippers. This is because in operation the centrifugal force associated with the stripper tends to force portions of the stripper outwardly as the stripper exits the nip between the cutting die and the anvil. By selectively adding additional weight in certain areas of the stripper, the positive effect of this centrifugal force is substantially increased.

The selective weighting of the scrap strippers can be carried out in a number of ways. For example, the material making up the scrap stripper may comprise a dual durometer material, meaning that one portion of the stripper would be more dense and consequently heavier than another portion. For example, in the case of the design shown in Figure 2, the entire flexible finger 22 or a portion thereof could be selectively weighted by a strip of material having a heavier density than the material making up other portions of the stripper. Another example of weighting the stripper of Figure 2 would entail placing small weighted pellets or balls in the outer terminal end of the flexible finger. The same approach would apply to the stripper 78 shown in Figure 5. The net effect of selectively weighting the stripper is that the portion of the stripper that engages the cut scrap

would impart additional force to the cut scrap and consequently would eject and control the trajectory of the cut scrap more efficiently.

The present invention may, of course, be carried out in other specific ways than those herein set forth without parting from the spirit and essential characteristics of the invention. The present embodiments are, therefore, to be considered in all respects as illustrative and not restrictive, and all changes coming within the meaning and equivalency range of the appended Claims are intended to be embraced therein.

065040" 99545060